

SKF 3143

Process Control and Dynamics: Response of Second Order Systems

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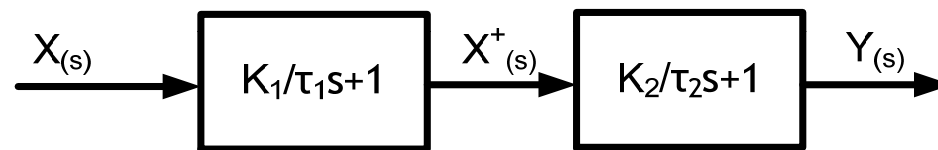
Learning Objectives

When I complete this chapter, I want to be able to do the following:

1. Analyze the response of the second order systems

Second-Order Systems

- A second-order transfer function can arise physically whenever two first-order processes are connected in series.
- For example, two stirred-tank heaters, each with first-order transfer function relating inlet to outlet temperature, might be physically connected so the outflow stream of the first heater is used as the inflow stream of the second tank.



- Here

$$G_{(s)} = \frac{Y_{(s)}}{X_{(s)}} = \frac{K_1 K_2}{(\tau_1 s + 1)(\tau_2 s + 1)} = \frac{K}{(\tau_1 s + 1)(\tau_2 s + 1)} \quad (7.1)$$

- In this chapter we consider the case where the second-order transfer function has the standard form

$$G_{(s)} = \frac{K}{\tau^2 s^2 + 2\zeta\tau s + 1} \quad (7.2)$$

Where ζ (zeta)= damping coefficient (dimensionless)

Second-Order Systems

- The type of behavior that occur depends on the numerical value of damping coefficient, ζ :
It is convenient to consider three types of behavior:

<i>Damping Coefficient</i>	<i>Types of Response</i>	<i>Root of Characteristic Polynomial</i>
$\zeta > 1$	Overdamped	Real and unequal
$\zeta = 1$	Critically damped	Real and equal
$0 < \zeta < 1$	Underdamped	Complex conjugates

- What about $\zeta < 0$? It results in unstable system
- The characteristic polynomial is the denominator of the transfer function:

$$\tau^2 s^2 + 2\zeta\tau s + 1$$

Second-Order Systems

- The transfer function given by Eqs. 7.1 and 7.2 differ only in the denominators.
- Equating them yields the relation between the two alternative forms for the *overdamped* second-order case.
- Note that when $\zeta \geq 1$, the denominator of 7.2 can be factored as:

$$\tau^2 s^2 + 2\zeta\tau s + 1 = (\tau_1 s + 1)(\tau_2 s + 1) \quad (7.3)$$

- Expanding the right side of 7.3 and equating coefficients of the s terms, yields

$$\tau^2 = \tau_1 \tau_2$$

$$2\zeta\tau = \tau_1 + \tau_2$$

from which we obtain

$$\tau = \sqrt{\tau_1 \tau_2}$$

$$\zeta = \frac{\tau_1 + \tau_2}{2\sqrt{\tau_1 \tau_2}}$$

Second-Order Systems

- Alternatively, the left side of 7.3 can be factored:

$$\tau^2 s^2 + 2\zeta\tau s + 1 = \left(\frac{\tau s}{\zeta - \sqrt{\zeta^2 - 1}} + 1 \right) \left(\frac{\tau s}{\zeta + \sqrt{\zeta^2 - 1}} + 1 \right) \quad (7.4)$$

from which expression for τ_1 and τ_2 are obtained

$$\tau_1 = \frac{\tau}{\zeta - \sqrt{\zeta^2 - 1}} \quad (\zeta \geq 1) \quad (7.5)$$

$$\tau_2 = \frac{\tau}{\zeta + \sqrt{\zeta^2 - 1}} \quad (\zeta \geq 1) \quad (7.6)$$

Response of Second-Order Systems

Step Response

- For the step input with transform:

$$Y_{(s)} = \frac{KM}{s(\tau^2 s^2 + 2\zeta\tau s + 1)} \quad (7.7)$$

- After some manipulation, and inverting to the time domain, three forms of response are obtained:

Case a ($\zeta > 1$)

If the denominator of Eq. 7.7 is factored using Eqs. 7.5 and 7.6, then the response can be written

$$y_{(t)} = KM \left(1 - \frac{\tau_1 e^{-t/\tau_1} - \tau_2 e^{-t/\tau_2}}{\tau_1 - \tau_2} \right) \quad (7.8)$$

If the denominator of Eq. 7.7 is left unfactored, then the response can be written in the equivalent form

$$y_{(t)} = KM \left\{ 1 - e^{-\zeta t/\tau} \left[\cosh \left(\frac{\sqrt{\zeta^2 - 1}}{\tau} t \right) + \frac{\zeta}{\sqrt{1 - \zeta^2}} \sinh \left(\frac{\sqrt{\zeta^2 - 1}}{\tau} t \right) \right] \right\} \quad (7.9)$$

Response of Second-Order Systems

Case b ($\zeta = 1$)

$$y(t) = KM \left[1 - \left(1 + \frac{t}{\tau} \right) e^{-t/\tau} \right] \quad (7.10)$$

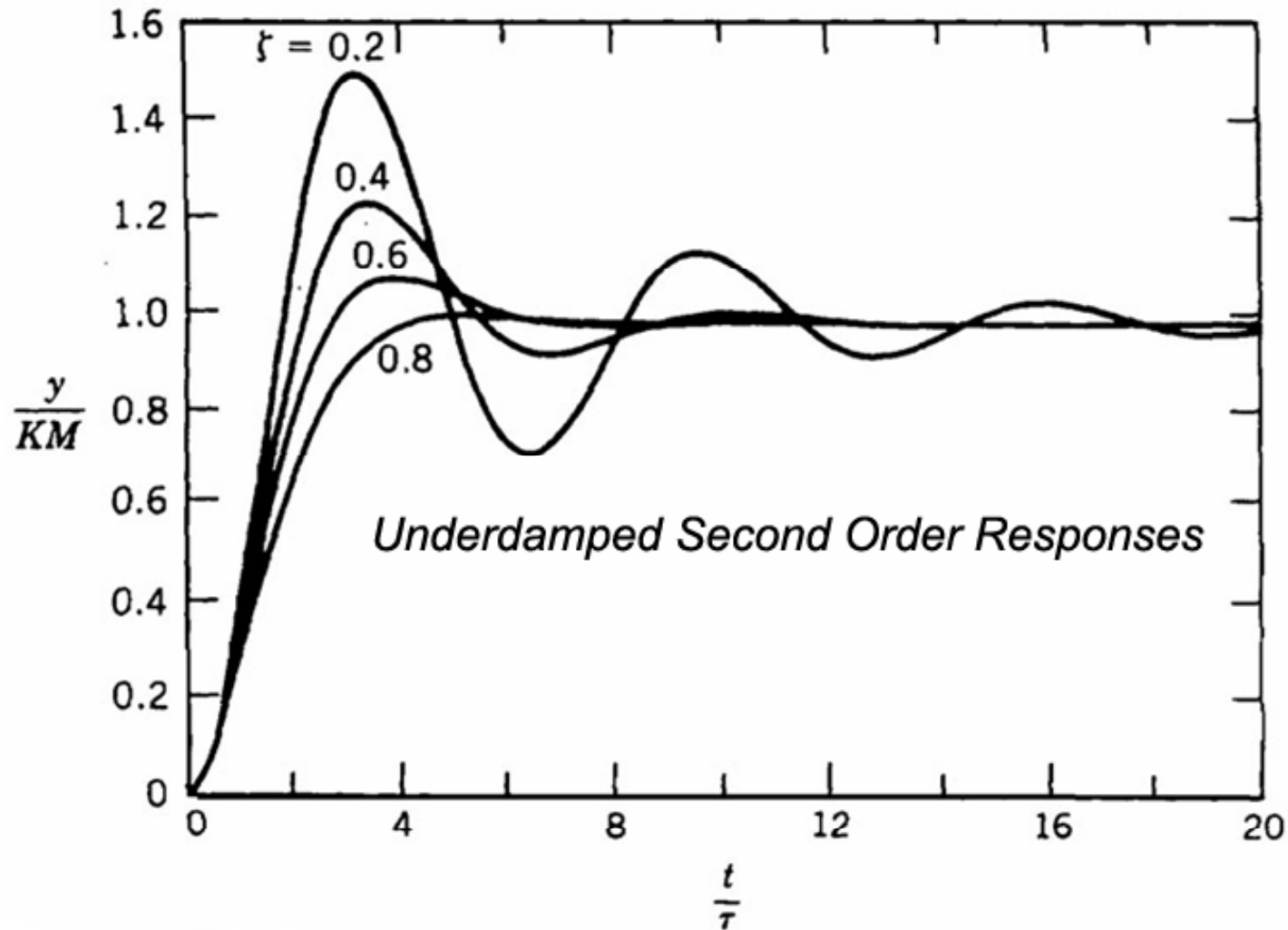
Case c ($0 \leq \zeta < 1$)

$$y(t) = KM \left\{ 1 - e^{-\zeta t/\tau} \left[\cos \left(\frac{\sqrt{1-\zeta^2}}{\tau} t \right) + \frac{\zeta}{\sqrt{1-\zeta^2}} \sin \left(\frac{\sqrt{1-\zeta^2}}{\tau} t \right) \right] \right\} \quad (7.11)$$

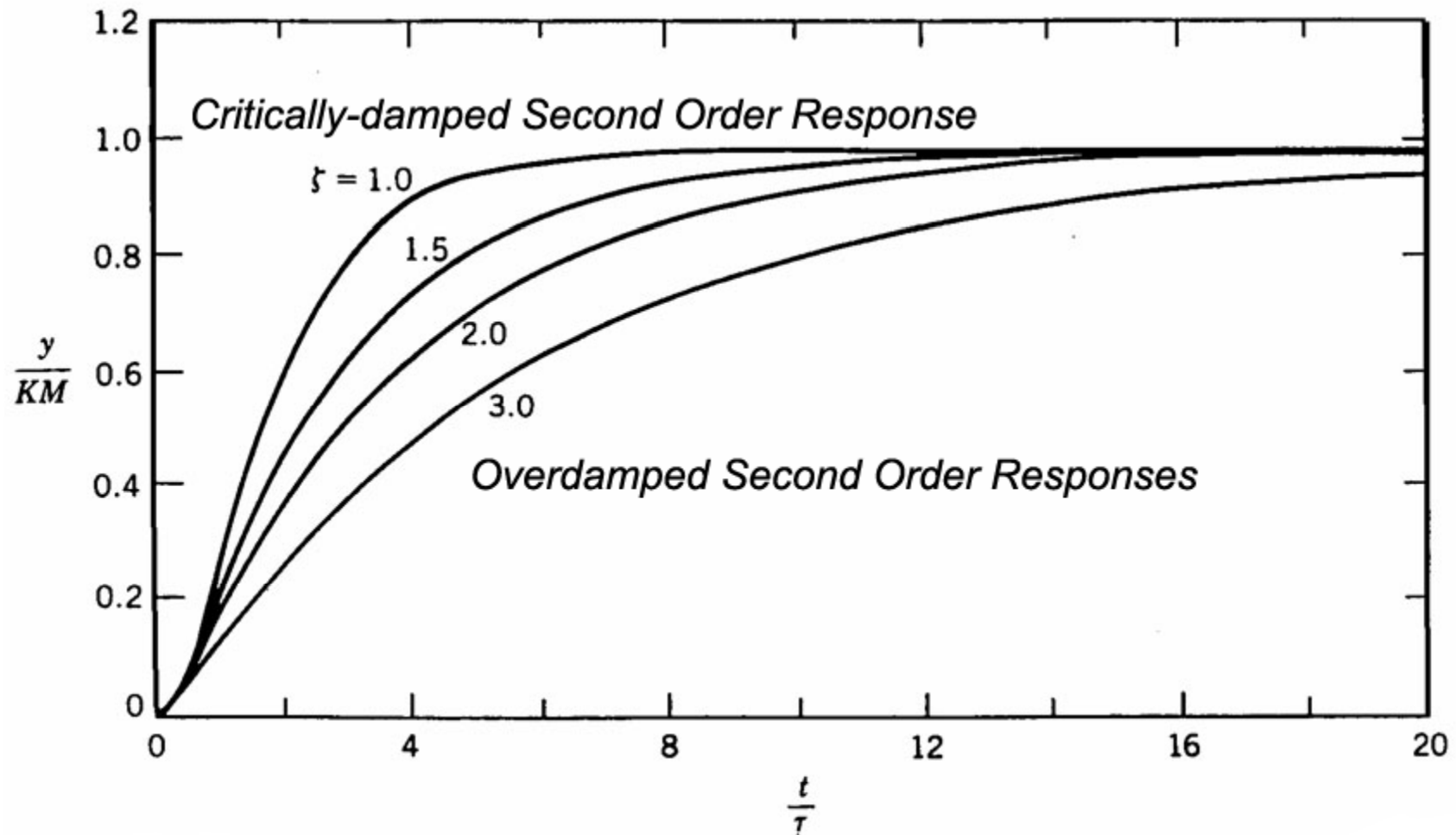
Several general remarks can be made concerning the responses shown in Figs. 7.1 and 7.2:

1. Responses exhibiting oscillation and overshoot ($y/KM > 1$) are obtained only for values of ζ less than one.
2. Large values of ζ yield a sluggish (slow) response.
3. The fastest response without overshoot is obtained for the critically damped case ($\zeta = 1$).

Step response of underdamped second-order systems



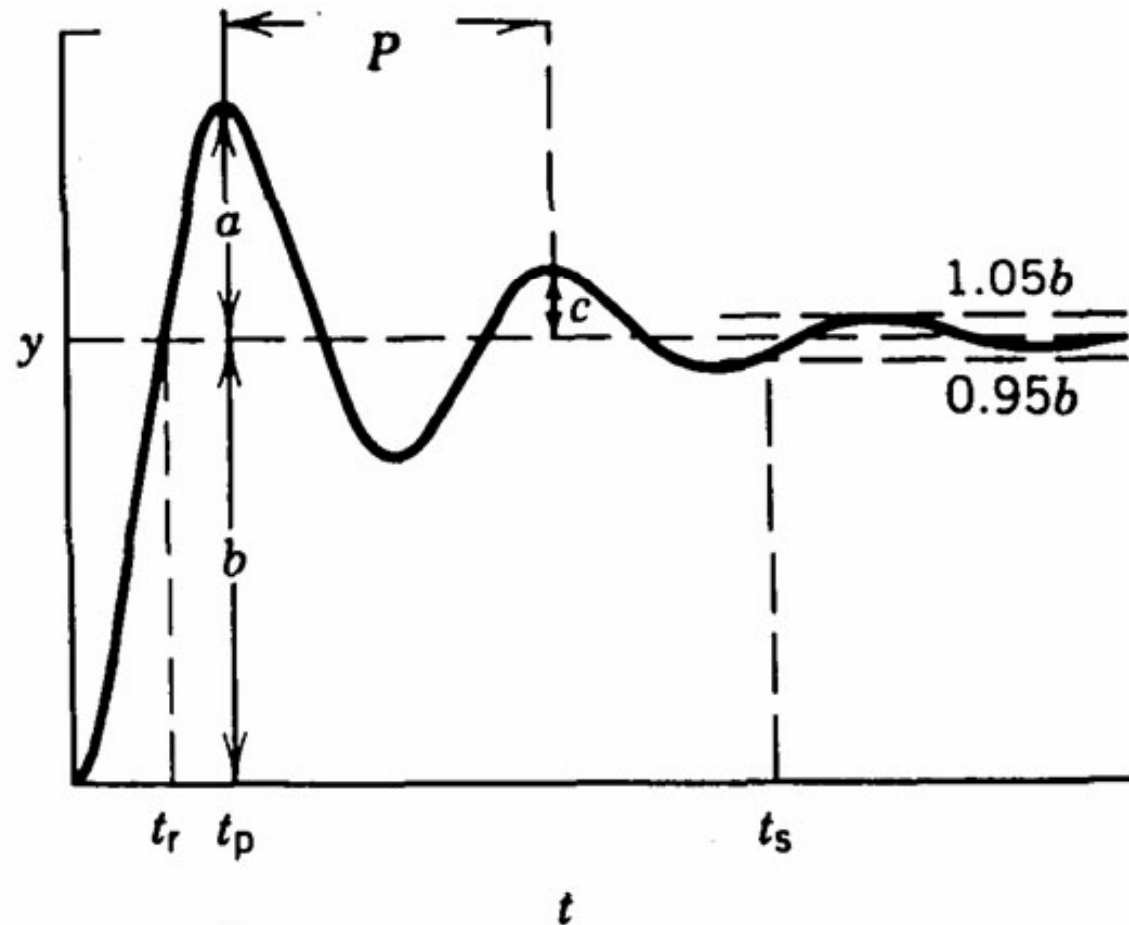
Step response of critically-damped and overdamped second-order systems



Response of Second-Order Systems

- Control system designers often attempt to make the setpoint step response of the controlled variable approximate the step response of underdamped second-order system.
- That is, make it exhibit a prescribed amount of overshoot and oscillation as it settles at the new operating point.
- Values of ζ in the range 0.4 to 0.8 often are suitable for specifying a desired control system response, assuming that it can be approximated as an underdamped second-order system.
- In this range, the controlled variable y reaches the new operating point faster than with $\zeta = 1$ or 1.5, but the response is much less oscillatory (it settles faster) than with $\zeta = 0.2$.

Performance characteristics for the step response of an underdamped process.



Underdamped Second Order Response

Response of Second-Order Systems

1. **Rise Time.** t_r is the time the process output takes to first reach the new steady-state value.
2. **Time to First Peak.** t_p is the time required for the output to reach its first maximum value.
3. **Settling Time.** t_s is defined as the time required for the process output to reach and remain inside a band whose width is equal $\pm 5\%$ of the total change in y . The term 95% response time sometimes is used to refer to this case. Also, values of $\pm 1\%$ sometimes are used.
4. **Overshoot.** $OS = a/b$ (% overshoot is $100a/b$).
5. **Decay Ratio.** $DR = c/a$ (where c is the height of the second peak).
6. **Period of Oscillation.** P is the time between two successive peaks or two successive valleys of the response.

References:

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